

MAGNETIC HEAD APPARATUS, MAGNETIC HEAD SUPPORTING  
MECHANISM AND MAGNETIC RECORDING APPARATUS

BACKGROUND OF THE INVENTION

5 Field of the Invention

The present invention relates to a magnetic head apparatus, magnetic head supporting mechanism and a magnetic recording apparatus. Specifically, the present invention relates to a magnetic head apparatus, magnetic head supporting mechanism and a magnetic recording apparatus that have improved impact resistance.

10 Related Background Art

Fig. 15 is a drawing schematically illustrating the outline of a prior art magnetic recording apparatus. The magnetic recording apparatus 1 shown in Fig. 15 is provided with a magnetic disk 2 serving as a rotatable recording medium and a magnetic head supporting mechanism 4 for moving a magnetic head 3 floating above the magnetic disk 2 in the radial direction of the magnetic disk 2. In the magnetic recording apparatus 1 having the above-described structure, a servo signal (i.e. position information) that has been written in the surface of the magnetic disk in advance is read by the magnetic head 3, and electric power is supplied to a movable coil 5 provided at the opposite end of the magnetic head 3

in accordance with the read information, so that a force is generated in a magnetic circuit 6 in the directions indicated by arrow 7. Thus, the magnetic head 3 is moved to a target track (or a target 5 position).

Fig. 16 is a drawing schematically showing how the magnetic head apparatus is disposed in relation to the magnetic disk. As shown in this drawing, a load beam 8 is provided at the intermediate portion 10 of the magnetic head 3. One end portion of the load beam 8 is secured to a base plate 9 that makes a junction with the magnetic head supporting mechanism 4. On the other end portion of the load beam 8, there is provided a slider 10 secured thereto. In 15 addition, a leaf spring portion is provided at the boundary 11 of the load beam 8 and the base plate 9. A pressing load (i.e. so-called load pressure) of the slider 10 against the magnetic disk 2 is provided by an urging force generated by this leaf spring portion.

20 However, the above-described magnetic recording apparatus suffers from the following problem. The mounting structure of the conventional magnetic head apparatus is a cantilever structure with the base plate 9 being the pivot. Therefore, if for example, 25 an impact is applied to it in the vertical direction (i.e. the thickness direction of the magnetic disk 2), a rotation moment (or torque) about the base plate 9

is generated with the slider 10 being a mass point. When the rotation moment thus generated exceeds the rotation moment with respect to the base plate 9 generated by the pressing load for the slider 10, the 5 slider 10 would detach from the surface of the magnetic disk 2 for a moment and then hit the surface of the magnetic disk 2. This can damage the magnetic head 3 itself or make a flaw on the surface of the magnetic disk 2 to deteriorate data that have already 10 been written.

On the other hand, the apparatus is so adapted that the pressing load for the slider 10 about the base plate 9 is generated by the leaf spring portion formed at the root side end of the load beam 8 (i.e. 15 at the boundary with the base plate 9). Therefore, it is required to form two different portions (i.e. the leaf spring portion and a rigid body portion) having different properties in the load beam 8, namely, the structure of the load beam 8 is complex. 20 This is another problem. In addition, forming of the leaf spring portion requires high precision bending processing on the load beam and inspection after the processing, which increases the number of the manufacturing processes. This is also a problem.

25 Various technologies for eliminating the above-mentioned problems have been proposed. For example, Japanese Patent Application Laid-Open No. 9-82052

discloses a structure provided with a second load beam that extends from a load beam attached with a slider at the opposite end thereof and a loading member provided on the second load beam so that the 5 center of the acceleration caused by an impact would coincide with the center of rotation of the slider.

Another document Japanese Patent Application Laid-Open No. 8-102159 discloses a structure in which a free end portion of a suspension can be in contact 10 with a pin-like projection provided on a base or a cover. In addition, Japanese Patent Application Laid-Open No. 2001-57032 discloses a structure provided with a limiter that is formed as an extension from a part of a base portion for mounting 15 a suspension to limit a movable range of the load beam so as to prevent a damage caused by an impact.

However, in the structure disclosed in Japanese Patent Application Laid-Open No. 9-82052, the load applied to the slider is given by a spring bias 20 provided in the load beam. Therefore, high precision bending processing is required to be made on the load beam. In addition, since a spring mechanism is present in the intermediate portion of the mechanism, it is difficult to prevent flipping due to a rotation 25 moment generated by an acceleration applied to the load beam. On the other hand, the structure disclosed in Japanese Patent Application Laid-Open No.

8-102159 provides a countermeasure only against an impact applied under the state in which the magnetic head apparatus is in a shipping zone (i.e. the state in which the magnetic disk is out of operation), but

5 it does not provide any countermeasure against an impact applied under the state in which the magnetic head apparatus is in the data zone (i.e. the state in which the magnetic disk is under operation). In addition, in the structure disclosed in Japanese

10 Patent Application Laid-open No. 2001-57032, in spite of the provision of the limiter for limiting the movable range of the load beam, the load applied to the slider is given by a spring bias provided in the load beam. Therefore, high precision bending

15 processing is required to be made on the load beam, as is the case with the structure disclosed in Japanese Patent Application Laid-Open No. 9-82052.

Furthermore, it would be conceived effective to shorten the distance from the leaf spring portion to

20 the slider by cutting down the whole length of the load beam for the purpose of reducing the moment of the slider as a mass point generated about the leaf spring portion. It is true that this countermeasure is effective for magnetic recording apparatus with

25 small media. However, in a normal magnetic recording apparatus having a medium with the size from 2.5 inches to 3.5 inches, it is necessary to extend the

length of the magnetic head supporting mechanism in order to compensate the cut down of the length of the load beam (since the medium size is large).

Therefore, there is a problem that it would be

5 difficult to realize an appropriate weight balance with respect to the rotation axis of the magnetic head supporting mechanism.

On the other hand, in a normal magnetic recording apparatus having medium with the size from

10 2.5 inches to 3.5 inches, if the load beam is elongated in order to reduce the moment of inertia of the magnetic head supporting mechanism about its rotation axis, the distance from the leaf spring portion to the slider is also elongated and the

15 weight (or mass) suspended by the spring (i.e. the weight of the load beam and the parts attached thereto such as the slider) increases due to the elongation of the load beam. Therefore, the slider becomes apt to be detached from the surface of the

20 recording medium when an impact is applied, and the impact resistance is reduced still more.

On the other hand, in order to realize a track seeking operation on the magnetic recording medium, it is necessary to provide a support member for a

25 head gimbal assembly. This support member, which is referred to as an arm, is constructed as a part extending from a pivot bearing portion in the

direction toward the medium. In view of spatial requirements in the interior of the magnetic recording apparatus, the supporting arm is generally constructed by a thin plate made of aluminum or a 5 stainless steel. However, such a thin plate does not have sufficient strength against an impact that may be applied to it, and therefore the support arm can deform at its free end when acceleration is generated by an impact. This sometimes causes a crash of the 10 head assembly attached on the tip of the arm.

#### SUMMARY OF THE INVENTION

The present invention has been made in view of the above-described problems. An object of the 15 present invention is to enhance impact resistance of a magnetic recording apparatus during both operating state and non-operating (or resting) state irrespective of the size of the apparatus and to provide a magnetic head apparatus, a magnetic head 20 supporting mechanism and a magnetic recording apparatus in which a pressing load against a recording medium can be set easily and with high precision.

The present invention has been made based on 25 the concept that in a magnetic head apparatus, a magnetic head supporting mechanism and magnetic recording apparatus, if the whole of a suspension is

constructed as a structure that behaves as an integral rigid body and a spring structure is provided between that rigid body and a mounting member, it is possible to provide a necessary 5 pressing load while a load caused by an impact is concentrated to the fulcrum of a balance structure.

According to the present invention, there is provided a magnetic head apparatus comprising a load beam to which a floating type slider is attached, an 10 elastically deformable portion provided on the load beam so that a floating structure that allows the load beam to swing is formed about the elastically deformable portion, and a load generating portion, wherein a position of the load generating portion is 15 adapted to coincide with a center of mass of said load beam, and a pressing load of the slider against a recording medium is set by a pressure generated at the load generating portion.

According to another mode of the present 20 invention, there is provided a magnetic head apparatus comprising a load beam to which a floating type slider is attached, an elastically deformable portion provided on the load beam, so that a floating structure that allows the load beam to swing is formed about the elastically deformable 25 portion, a projecting portion for generating a load disposed in the vicinity of the elastically

deformable portion of the load beam, and a pressure receiving surface provided on said load beam for receiving a pressure from said projecting portion, wherein a position of said projecting portion for generating a load is adapted to coincide with a center of mass of said load beam, and a pressing load of said slider against a recording medium is set by a pressure applied to said pressure receiving surface.

According to a more specific mode of the present invention, there is provided a magnetic head apparatus comprising a base plate adapted to be attached to a head arm, a load beam that extends from the base plate, a floating type slider attached to the load beam, an elastically deformable portion provided between the base plate and the load beam, so that a floating structure that allows said load beam to swing is formed about the elastically deformable portion, a projecting portion for generating a load disposed in the vicinity of the elastically deformable portion of the load beam, a pressure receiving surface provided on the load beam, wherein a position of said projecting portion for generating a load is adapted to coincide with a center of mass of said load beam, a pressing load is applied to a surface of a recording medium via the floating type slider, and a pressing load of the slider against the recording medium is set by a

pressure applied to said pressure receiving surface. The balancing of the weight may be attained by providing a dead weight in the form of an ordinary balance weight or a vibration damping member or using 5 other functional parts. Furthermore, it is preferable that the load beam be made of a lightweight metal such as a stainless steel or aluminum or a lightweight material such as a resin. As the resin for the load beam, an electrically 10 conductive resin may be used in order to attain electrical contact with an external member. Alternatively, an electrically conductive coating may be formed on the resin in order to attain electrical contact with an external member via the electrically 15 conductive coating. The above-mentioned dead weight may be made of a resin. In addition, the head arm may be supported in such a way as to be pivotable in a radial direction of said recording medium, and the head arm may have a strengthen plate that is attached 20 to said head arm perpendicularly in such a way that it would not interfere with said recording medium.

According to the present invention, there is also provided a magnetic head supporting mechanism comprising a magnetic head apparatus including a base 25 plate and a load beam extending from the base plate, a head arm attached to said base plate, an elastically deformable portion that is flexible

provided between the base plate and the load beam so that a floating structure that allows the load beam to swing is formed about the elastically deformable portion, a projecting portion for generating a load  
5 disposed in the vicinity of the elastically deformable portion of the load beam, and a projecting portion for generating a load provided on the head arm, the projecting portion being adapted to apply a pressure to the load beam, wherein a position of the  
10 projecting portion for generating a load is adapted to coincide with a center of mass, a pressing load is applied to a recording medium via a floating type slider attached to the load beam, and the pressing load to the recording medium is set by an amount of  
15 rotation of the load beam caused by the pressure applied by the projecting portion for generating a load. The position of the above-mentioned center of mass may be adjusted by using a dead weight made of a vibration damping member or other functional parts as  
20 well as an ordinary weight. Furthermore, it is preferable that the load beam be made of a lightweight metal such as a stainless steel or aluminum or a lightweight material such as a resin. As the resin for the load beam, an electrically  
25 conductive resin may be used in order to attain electrical contact with an external member. Alternatively, an electrically conductive coating may

be formed on the resin in order to attain electrical contact with an external member via the electrically conductive coating. The above-mentioned dead weight may be made of a resin. In addition, the head arm

5 may be supported in such a way as to be pivotable in a radial direction of said recording medium, and the head arm may have a strengthen plate that is attached to said head arm perpendicularly in such a way that it would not interfere with said recording medium.

10 The base plate may be provided as a separate member different from the load beam, or alternatively, the base plate may be provided as a part integral with the load beam.

Furthermore, according to the invention, there

15 is provided a magnetic recording apparatus equipped with the above-mentioned magnetic head apparatus or magnetic head supporting mechanism.

In connection with the above, the term "floating structure" refers to a structure in which a

20 load beam and a base plate is not joined by a rigid body, so that an impact applied on the base plate can prevented from being transmitted to the load beam linearly.

In the above-described structures of the

25 various apparatus, the elastically deformable portion is provided at the position at which the load beam attached with the slider is balanced. (Alternatively,

the elastically deformable portion is provided on the load beam, and a dead weight is attached to the load beam at a position opposite to the position at which the slider is attached so that the balancing would be

5 attained.) When a floating structure that allows the load beam to swing is realized by the support or suspension of the load beam by the elastically deformable portion positioned as above, it is possible to prevent a rotational force about the

10 elastically deformable portion from being created, even if an impact is applied to the load beam. Thus, the slider will not be detached from the recording medium by such a rotational force. Therefore, it is possible to prevent the slider from colliding with

15 the recording medium to damage it or prevent the magnetic head apparatus itself from being damaged by an impact. The load beam would be rotated or swung about the elastically deformable portion by application of an external pressure to the load beam.

20 Therefore, a pressing load of the slider against the recording medium can be set (or determined) by adjusting the amount of the rotation. Since the pressing load is determined by the rotation amount of the load beam in this way, it is possible to create

25 an accurate pressing load and to suppress variations in the pressing load. In addition, an elastic portion for providing a pressing load to the load

beam need not be formed on the load beam, and so a high precision bending process for processing the load beam or an inspecting process for measuring a spring load can be omitted. Therefore, it is  
5 apparent that the manufacturing process of the apparatus can be made simple.

With the structure in which the load beam is joined to the base plate via the elastically deformable portion, it is not necessary to construct  
10 the whole of the magnetic head apparatus or the whole of an actuator (including a head arm and VCM etc.) as a floating structure, but only the load beam and the parts attached thereto on the tip portion of the magnetic head apparatus would be constructed as a  
15 floating structure. Therefore, the weight or mass of the portion provided below the elastically deformable portion is reduced (i.e. reduction of the mass suspended by a spring). This reduction of the weight would results in enhancement of impact resistance.

20 Furthermore, with the structure in which a contact portion is formed on the head arm, when the pressure receiving surface on the load beam is pressed by the contact portion, the load beam is rotated about the elastically deformable portion by  
25 an amount corresponding to the height of the contact portion. Therefore, it is possible to obtain a pressing load without a variation between individual

products, by controlling the projecting height of the contact portion.

The balancing of the weight of the load beam with respect to the elastically deformable portion 5 may be attained by adding a dead weight to the load beam and/or by forming a hole for the purpose of reducing the weight. When the dead weight is attached to the load beam, a vibration damping member such as a vibration suppressing steel plate may 10 preferably be used as the dead weight. In that case, the peak value of the natural resonance frequency of the load beam can be reduced as desired. Therefore, stability of the actuator system can be enhanced.

In the present invention, the load beam is not 15 required to have an elastic portion, and therefore the load beam may be made of various materials. In other words, the material of the load beam is not limited to conventional metal materials such as a stainless steel, but the load beam may be made of a 20 resin. With the use of load beam made of a resin, it is possible to reduce the weight greatly as compared to conventional load beams made of metal materials. In other words, the weight or mass of the portion provided below the elastically deformable portion is 25 reduced with the use of the load beam made of a resin (i.e. reduction of the mass suspended by the spring). This reduction of the weight would further enhance

impact resistance.

If the resin for the load beam is an electrically conductive resin, it is possible to make the electric potential of the load beam, the actuator and the base side of the magnetic recording apparatus equal to each other. Therefore, it is possible to prevent electrostatic discharge from occurring on the load beam. Thus, damaging of the magnetic head apparatus by static electricity can be prevented.

10 The same effect would also be realized by forming an electrically conductive coating on a resin instead of using an electrically conductive resin. It is preferable that the electrically conductive coating be a metal coating in view of its low volume

15 resistance. It is apparent that the combination of an electrically conductive resin and an electrically conductive coating would realize a more preferable effect.

With the use of the above-described magnetic head apparatus or the actuator in a magnetic recording apparatus, it is possible to enhance impact resistance of the magnetic recording apparatus both in the operating state and in the non-operating state, irrespective of the size of the magnetic recording apparatus. Therefore, reliability of the magnetic recording apparatus can be enhanced.

25

It should be understood that the term "magnetic

head apparatus" refers to an apparatus in the form of a head gimbal assembly (HGA) including a slider and a load beam, while the term "magnetic head supporting mechanism" refers to a structure including the

5 magnetic head apparatus and a head arm (and a base plate).

Other features and objects of the present invention will become apparent from the following detailed description and the annexed drawings.

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#### BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a plan view showing the structure of a magnetic head apparatus as an embodiment of the present invention.

15 Fig. 2 is a drawing for illustrating states in which a load beam is swung by a pressing force applied to a pressure receiving surface.

Fig. 3 is an exploded view showing how the magnetic head apparatus and a head arm are assembled

20 in relation to each other.

Fig. 4 is a front view showing a magnetic head supporting mechanism formed by mounting the magnetic head apparatus to the head arm.

Fig. 5 is a drawing for illustrating how the

25 magnetic head apparatus according to the embodiment of the present invention is assembled with a recording medium, which shows a state before the

assembling.

Fig. 6 is a drawing for illustrating how the magnetic head apparatus according to the embodiment is assembled with a recording medium, which shows a 5 state after the assembling.

Fig. 7 is a plan view showing a magnetic recording apparatus equipped with the magnetic head or a magnetic head supporting mechanism according to the embodiment of the present invention.

10 Fig. 8 is a cross sectional view taken on line 8-8 in Fig. 7.

Fig. 9 is a schematic drawing for illustrating impact resistance performance of the magnetic head apparatus according to the embodiment of the present 15 invention.

Fig. 10 is an exploded view showing a modification of the magnetic head supporting structure according to the embodiment of the present invention.

20 Fig. 11 is a side view showing an arm to which strengthen plates are attached.

Fig. 12 is a plan view showing the arm assembly to which strengthen plates are attached.

Fig. 13 is a drawing showing an arm assembly 25 including multiple heads to which strengthen plate are attached.

Fig. 14 is a drawing showing an arm assembly

including a single head to which a strengthen plate is attached.

Fig. 15 is a drawing illustrating the outline of a conventional magnetic recording apparatus.

5 Fig. 16 is a drawing schematically illustrating how a magnetic head apparatus is assembled with a magnetic disk.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

10 In the following embodiments of a magnetic head, a magnetic head supporting mechanism and a magnetic recording apparatus according to the present invention will be specifically described with reference to the annexed drawings.

15 Fig. 1 is a plan view showing the structure of a magnetic head apparatus as an embodiment of the present invention.

As shown in Fig. 1, the magnetic head apparatus 20 according to this embodiment has a load beam 22 having an outer shape like an isosceles triangle. On an inner portion of the load beam 22, there is provided a base plate 24 functioning as a mount portion to be attached to a head arm (which will be described later).

25 The load beam 22 is made by pressing or etching a thin metal plate. More specifically, the thin metal plate is a non-magnetic stainless steel (e.g.

an austenite stainless steel). The edges of the load beam 22 corresponding to the two isometric sides of the isosceles triangle shape are formed into bent portions 26. Each bent portion 26 is formed by

5 bending the edge of the load beam 22 at a certain angle or bending the edge into a half round shape (i.e. semicylinder shape). With the provision of the bent portions 26, rigidity with respect to the longitudinal direction of the load beam 22 can be

10 assured.

At the center of the load beam 22 between the bent portions 26 formed at the right and left edges, a slit 28 of an inverted U-shape (in Fig. 1) is formed. The tongue surrounded by the slit 28 is

15 adapted to serve as the above-mentioned base plate 24.

The boundary portion between the base plate 24 and the load beam 22 (namely, the portion along line 30 in Figs. 1 and 2) functions as a cantilevered leaf spring portion that serves as an elastically

20 deformable portion. At positions slightly offset from line 30 on the load beam 22, there is provided a pair of pressure-receiving surfaces 34. Thus, after the base plate 24 is fixed, the load beam 22 can swing or pivot about line 30 upon receiving pressing

25 force applied externally of the magnetic head apparatus 20 on the pressure receiving surfaces 34. The swinging of the load beam 22 in response to

application of a pressing force is illustrated in Fig. 2.

At the tip end portion (i.e. the upper end portion in Fig. 1) of the load beam 22, a slider 36 (see Fig. 3) in which an element for performing writing/reading of a recording medium is assembled is mounted via a gimbal (not shown).

Fig. 3 is an exploded view showing how the magnetic head apparatus and a head arm are assembled in relation to each other. Fig. 4 is a front view of a magnetic head supporting mechanism formed by mounting the magnetic head apparatus to the head arm.

As shown in Figs. 3 and 4, at the tip end portion of the head arm 38 to which the magnetic head apparatus is to be mounted, there is provided a plate mounting surface 40 to which the base plate 24 is to be fixed. The size of the plate mounting surface is substantially the same as the size of the base plate 24 in the magnetic head apparatus. The head arm 38 has a recess 42 formed at the periphery of the plate mounting surface 40. The recess 42 has the width sufficient for receiving the width of the load beam 22, so that when the magnetic head apparatus is assembled with a recording medium, the rear end portion of the load beam 22 is prevented from interfering with the head arm 38. If the magnetic head apparatus in a floating (or hovering) state does

not obstruct loading, the recess 42 may be omitted.

The head arm 38 is further provided with a pair of projections serving as contact portions formed at positions that are closer to the tip end than the 5 plate mounting surface 40. When the base plate 24 is aligned with the plate mounting surface 40, the top portions of the pair of projections 44 are in contact with the pressure receiving surfaces 34 provided on the load beam in advance to press them.

10 The head arm is provided with a center hole 46 in which a bearing is accommodated and a coil 48 constituting a VCM (i.e. voice coil motor) disposed on the rear side of the center hole 46. The head arm 38 can swing about the center hole 46 with supply of 15 electrical power to the coil 48. It is desirable that the magnetic head supporting mechanism 50 including the magnetic head apparatus 20, the head arm 38 and the coil 48 be balanced with respect to the center hole 46, in order to minimize influences 20 of external disturbances.

Figs. 5 and 6 are drawings illustrating how the magnetic head apparatus according to the present embodiment is assembled to the recording medium.

As shown in Fig. 5, the magnetic head apparatus 25 20 according to the present embodiment is first fixed to the head arm 38 by spot welding or other attaching processes. When the magnetic head apparatus 20 is

fixed to the head arm 38, the pair of projections 44 provided at the tip end portion of the head arm 38 press the load beam 22 to cause the load beam 22 to swing or pivot in such a way that the slider 36 is 5 lowered relative to the recording medium 52. In this process, the load beam 22 can swing without being flexed, since rigidity is assured by the bent portions 26 formed at both the edges. Even when the load beam 22 is swung by pressure applied by 10 projections 44, the rear end of the load beam 22 does not interfere with the head arm 38 by virtue of the presence of the recess 42 formed on the head arm 38. (In other words, the recess 42 should be formed with the depth that is sufficient for preventing the 15 interference in accordance with the inclination of the load beam 22.) Therefore, it is also possible to prevent dust from being generated by interference of the parts.

As shown in this drawing, after the magnetic 20 head is fixed to the head arm 38, the load beam is swung, by means of a jig, in such a way that the slider 36 comes to a position higher than the surface of the recording medium 52, and then the slider 36 is landed (or loaded) on the surface of the recording 25 medium 52. Fig. 6 shows the apparatus in this state. In the state shown in Fig. 6, the following condition is met, where A represents the distance from the

projections 44 for creating load to the junction of the leaf spring and the load beam, B represents the distance from the projections 44 to the slider 36, F1 is the pulling-up force of the leaf spring and F2 is 5 the reactive force exerted to the slider 36 by the recording medium 52 (a loss that might occur due):

$$F1 \cdot A = F2 \cdot B \quad (\text{conditional expression 1}).$$

In other words, the moment about the projections 44 created by the pressing force is equal to the moment 10 created by the reactive force. Therefore, the reactive force of the recording medium 52 that influences the floating characteristics of the slider can be set or adjusted by adjusting the height of the projections 44.

15 Fig. 7 is a plan view showing a magnetic recording apparatus equipped with the magnetic head or the magnetic head supporting mechanism according to the present invention. Fig. 8 is a cross sectional view taken at line 8-8 in Fig. 7.

20 The distinguishing features of the magnetic recording apparatus shown in these drawing reside in the magnetic head supporting structure 50 and its peripheral structures, and other parts of the apparatus such as a spindle motor for rotationally 25 driving the recording medium 52 are the same as those in conventional apparatus. Therefore, a magnetic recording apparatus 54 having improved impact

resistance can be realized only by substituting the magnetic head supporting structure 50 for that in a conventional apparatus.

Fig. 9 is a schematic drawing for illustrating 5 the impact resistance performance of the magnetic head apparatus according to the embodiment of the present invention.

As shown in Fig. 9, the head arm 38 and the load beam 22 are connected by the elastically 10 deformable portion 56, and the pressure receiving surface 60 of the load beam 22 is pressed by the contact portion 58 provided on the head arm 38. The weight of the magnetic head apparatus 20 is arranged to be balanced with respect to the projecting portion 15 58. The balancing of the weight may be realized by adjusting the position of the elastically deformable portion 56 on the load beam and/or attaching a dead weight 62 on the load beam 22 at a position opposite to the slider 36 as shown in Fig. 9. In addition, if 20 the dead weight 62 is made of a vibration damping member (or damper), it is possible to reduce the peak value of resonance with respect to the magnetic head apparatus 20, to thereby stabilize a control system 25 (for positioning etc.) of the magnetic recording apparatus 54.

By virtue of the balanced weight of the magnetic head apparatus 20 with respect to the

projecting portion 58, even if an impact is exerted on the apparatus in the direction of arrows 64 shown in Fig. 9, no rotational force is created in the load beam 22. Thus, the slider 36 is prevented from being 5 detached from the surface of the recording medium when a strong impact is applied. Therefore, it is possible to eliminate adverse effects such as damaging of elements assembled in the slider or a defect on the recording medium 52 formed by collision 10 with the slider 36.

In the apparatus according to this embodiment, since only the load beam 22 and the parts attached thereto (i.e. the slider 36 and the dead weight 62) are constructed as a floating structure via the 15 elastically deformable portion 56, the mass suspended by the spring (i.e. the total mass of the load beam 22 and the parts attached thereto) below the elastically deformable portion 56 would be reduced. Letting  $W$  be the mass of the load beam 22 and the 20 parts attached thereto suspended by the elastically deformable portion 56,  $F_s$  be a pressing force applied to the load beam 22 by the contact portion 58, and  $a$  be an impact acceleration created in the load beam 22 and the parts attached thereto, the following 25 relation is met:

$$F_s = W \cdot a. \quad (\text{conditional expression 2}).$$

The inventors estimated by calculation the degree of

improvement in impact resistance realized by the present invention. Assuming the mass  $W=30\text{mg}$  (milligrams) and  $F_s=120\text{g}$  (grams) the above relation is as follows:

5            $120=0.03 \cdot a$ , therefore  
               $a=4000$ .

This means that as far as the impact acceleration is smaller than 4000G, it is possible to prevent the load beam 22 from detaching from the loading  
10 projections 44, and therefore, the slider 36 can be prevented from detaching from and colliding with the recording medium. Thus, the impact resistance can be greatly enhanced as compared to conventional apparatus. In addition the impact resistance of the  
15 magnetic head apparatus 20 according to this embodiment does not depend on the length of the head arm, namely it does not depend on the size of the recording medium.

The material of the load beam 22 is not limited  
20 to the above-described thin metal plate, but other materials can also be used as long as rigidity is assured.

The inventors have discovered that a resin is also used as a material for the load beam 22 instead  
25 of the thin plate of a stainless steel that have been conventionally used. With the use of a resin for the load beam 22, the mass suspended by the spring

portion would be further reduced, and therefore the impact resistance performance can be improved still more. The inventors found that resins suitable for the load beam 22 are liquid crystal polymer resins or

5 PPS resins that have electric conductivity, in view of their ability of preventing ESD (i.e. electro static discharge). It is desirable that the specific volume resistance of these resins be smaller than  $10^5 \Omega \text{cm}$ .

10 Even a resin that does not have electric conductivity can also be effectively used if a metal coating is formed by plating etc. on the surface of the load beam 22 after it is injection molded so that its electrical potential would be always kept equal  
15 to the potential of the head arm 38.

Fig. 10 is an exploded view showing a modification of the magnetic head apparatus according to the above-described embodiment. In Fig. 10, parts having the same functions as the parts in the above-  
20 described embodiment are designated with the same reference numerals and descriptions thereof will be omitted.

In the structure of the modification shown in Fig. 10, the orientation of the slit and the spring portion is reversed and the projecting portion is formed as a part extending transversely to the load beam on the slider side of the base plate with the

spring portion between to create a load. In this case also, the weight or mass of a suspension slider assembly is arranged in such a way that its center of mass coincides with the projecting portion. In this 5 structure, a necessary load can be provided as long as the conditions of the above-described principles are met, and the structure has stability against an impact.

As per the above, the magnetic head apparatus, 10 the magnetic head supporting structure and the magnetic recording apparatus according to the above-described embodiments have improved impact resistance that does not depend on the size or the number of the recording medium(s).

15 While in the foregoing description of the embodiments the description has been made of a magnetic recording apparatus 54 of a CSS (contact start stop) type, the invention is not limited to the apparatus of that type. The invention may also be 20 applied to an apparatus of a ramp load type in which a tab is provided at the tip of the load beam 22, which allows the slider to be retracted from the surface of the recording medium when it is not operated. In the apparatus of the ramp load type, 25 when the apparatus is out of operation, the slider is riding on a ramp so that the slider and the recording medium are protected, while when the apparatus is

under operation the slider and the recording medium are protected by the structure according to the embodiment. Therefore, reliability of the magnetic recording apparatus can be greatly enhanced.

5       As described before, in order to realize a track seeking operation on the magnetic recording medium, it is necessary to provide a support member for a head gimbal assembly. This support member, which is referred to as an arm, is constructed as a  
10      part extending from a pivot bearing portion in the direction toward the medium. In view of spatial requirements in the interior of the magnetic recording apparatus, the supporting arm is generally constructed by a thin plate made of aluminum or a  
15      stainless steel. However, such a thin plate does not have sufficient strength against an impact that may be applied to it, and therefore the support arm can deform at its free end when acceleration is generated by an impact. This sometimes causes a crash of the  
20      head assembly attached on the tip of the arm. In order to solve this problem, a strengthen plate(s) is attached to a head arm assembly including one or more head arms in order to enhance the strength of the arms against impact acceleration. The strengthen  
25      plate is attached to a side of the head arm assembly other than the side facing the recording medium in such a way that the strengthen plate extends

perpendicular to that side of the arm assembly.

The arm has a projection(s) for generating the above-mentioned load disposed at a position at which a suspension is attached, wherein the impact  
5 resistance of the suspension itself has already been enhanced. With the provision of the strengthen plate(s) attached to extend perpendicularly to its side, the arm has a structure similar to a box that is supported at one side. Therefore, the strength  
10 against impact of the suspension attaching portion is enhanced. The arm(s) 72 to which the strengthen plate(s) 70 is attached is shown in Figs. 11 to 14.

As has been described in the foregoing, in a magnetic head apparatus according to the present  
15 invention, an elastically deformable portion is provided on a load beam to which a floating type (or hovering type) slider is attached so that a floating structure that allows the load beam to swing would be formed about the elastically deformable portion, a  
20 center of pressure disposed in the vicinity of the elastically deformable portion is adapted to coincide with a center of mass, a pressure receiving surface for receiving pressure from a projecting portion is provided on the load beam, and a pressing load of the  
25 slider against a recording medium is set by a pressure generated at the load generating portion. A magnetic recording apparatus according to another

mode of the present invention has a base plate and a load beam extending from the base plate, wherein a pressing load is applied to a surface of a recording medium via a floating type slider attached to the

5 load beam, an elastically deformable portion is provided between the base plate and the load beam so that a floating structure that allows the load beam to swing would be formed about the elastically deformable portion, a center of pressure formed in

10 the vicinity of the elastically deformable portion of the load beam is adapted to coincide with a center of mass, a pressure receiving surface for receiving pressure from a projecting portion is provided on the load beam, and a pressing load of the slider against

15 a recording medium is set by a pressure generated at the load generating portion.

Furthermore, a magnetic head supporting mechanism according to the present invention has a magnetic head apparatus provided with a base plate and a load beam extending from the base plate and a head arm attached to the base plate, wherein a pressing load is applied to a recording medium via a floating type slider attached to the load beam, an elastically deformable portion that is flexible is provided between the base plate and the load beam so that a floating structure that allows the load beam to swing would be formed about the elastically

deformable portion, the weight of the load beam is balanced with respect to the elastically deformable portion, and a contact portion for applying a pressure to the load beam is provided on the head arm 5 so that the pressing load to the recording medium is set by an amount of rotation of the load beam caused by the pressure applied by the contact portion.

With the above-described features, impact resistance performance of the apparatus can be 10 enhanced, and setting of pressing load to the recording medium can be effected easily with high accuracy. Therefore, it is possible to enhance reliability of the magnetic recording apparatus.